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Degree : B.I

Branch - Stream : MECHANICAL

Course Title: THEORY OF MACHINES

Duration : 60 Minutes

USN

Semester: V
Course Type / Code: THEORY/21ME51

Date: 2/1/2024

Max Marks: 20

Note: Answer ONE full question from each Module.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Mark s	со	K- Level
	Module-1			
1(a)	Define i) kinematic pair ii) kinematic chain iii)degree of freedom iv)machine	4	COI	K2
(p)	Make use of contact between the links and type of motion between the links to classification of kinematic pairs	8	CO1	КЗ
	OR			
2(a)	Define i)mechanism ii)inversion iii)lower pair & Higher pair iv)mechanism	4	COI	K2
(b)	In a 4 bar kinematic chain ABCD, AD is fixed and is 150mm long, the crank AB is 40mm long and rotates at 120 rpm CW, while the link CD-80mm oscillates about D. BC and AD are of equal length. Solve the angular velocity of link CD when BAD=60°	8	CO1	К3
	Module-2			
3(a)	Explain static equilibrium conditions for the following i) Body subjected to two forces ii) body subjected to three forces iii)Body subjected to two forces and torque	4	CO2	K2
(b)	The crank of slider crank mechanism rotates CW at a constant speed of 300 rpm. The crank is 150mm and connecting rod is 600mm ling. Determine linear velocity of the piston	4	CO2	кз
	OR			
4(a)	Explain tangential and radial component of acceleration.	4	CO2	К2
(b)	In a 4 bar mechanism ABCD, is made up of four links, pin jointed at the ends AD is fixed link, which is 180mm long, the links AB, BC, and CD are 90mm, 120mm and 120mm long respectively. At certain instant the link AB makes an angle of 60° with the link AD. If the link AB rotates at uniform speed of 100 rpm CW determine i) angular velocity of the link BC and CD	4	CO2	КЗ

Name & Signature of Course In charge M. Warlun

(M. Warlun (1909)

Name & Signature of

Module Coordinator

HOD

Principal Principal



SCHEME

SET: A

Branch - Stream :

Course Title:

Degree : B.E

MECHANICAL

MECHANICAL THEODY ODAY

THEORY OF MACHINES

Duration : 60 Minutes

USN | Semester : V

Course Type / Code: THEORY/21ME51

Date: 2/1/2024

Max Marks: 20

Note: Answer ONE full question from each Module.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Mark
1(a)	I)kinematic pair – 1mark ii) kinematic chain – 1mark iii)degree of freedom – 1mark iv)machine – 1mark	4
(b)	lower pair- 2marks higher pair – 2marks kinamtic pairs based on relative motion- 4marks	8
2(a)	i)mechanism- 1mark ii)inversion-1mark iii)lower pair & Higher pair -1mark iv)mechanism-1mark	4
(b)	4 bar kinematic chain space diagram- 2 mark velocity of the crank-2 marks velocity diagram- 2marks angular velocity of link CD 2marks	8
3(a)	Definition of static equilibrium -1 mark i) Body subjected to two forces - 1 mark ii) body subjected to three forces - 1 mark iii) Body subjected to two forces and torque- 1 mark	4
(b)	Slider crank mechanism space diagram- 2 mark velocity of the crank-2 marks velocity diagram- 2marks linear velocity of piston 2marks	4
4(a)	Tangential component of acceleration-2 marks Radial component of acceleration- 2marks	4
(b)	4 bar mechanism ABCD, space diagram- 2 mark velocity of the crank-2 marks velocity diagram- 2 marks angular velocity of the link BC and CD-2 marks	4

Name & Signature of Course In charge



SET: B

Degree : B.E

Branch - Stream : MECHANICAL Course Title : THEORY OF M

Duration :

THEORY OF MACHINES

: 60 Minutes

Semester : V

Course Type / Code: THEORY/21ME51

Date: 2/1/2024

Max Marks: 20

Note: Answer ONE full question from each Module.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions Questions	Mark s	со	K- Level
	Module-1			
1(a)	Differentiate between machine and mechanism	4	CO1	K2
(b)	Explain classification of kinematic inversions of 4 bar kinematic chain	8	CO1	К2
	OR	- 33	- 10	
2(a)	Define i)mechanism ii)inversion iii)Iowcr pair & Higher pair iv)DOF	4	CO1	КЗ
(b)	Explain the procedure to follow velocity and acceleration analysis of 4 bar kinematic chain with an example	8	COI	КЗ
	Module-2			
3(a)	Define static equilibrium and Explain static equilibrium conditions for the following i) Body subjected to two forces ii) body subjected to three forces	4	CO2	КЗ
(b)	Draw the FBD of all the movable links shown in the figure B F 160 C 45 / P AB = 50 mm CF = 30 mm BC = 66 mm BAO = 60 CD = 55 mm AD = 100 mm ACT	4	CO2	КЗ
	OR			
4(a)	Explain tangential and radial component of acceleration.	4	CO2	КЗ
(b)	In a 4 bar mechanism ABCD, is made up of four links, pin jointed at the ends AD is fixed link, which is 180mm long, the links AB, BC and CD are 90mm, 120mm and 120mm long respectively. At certain instant the link AB makes an angle of 60° with the link AD. If the link AB rotates at uniform speed of 100 rpm CW determine i) angular velocity of the link BC abd CD	4	CO2	КЗ

Name & Signature of Course In charge (IN. WALL BUSINAM)
Name & Signature of
Module Coordinator

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Principal Principal

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SCHEME

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Degree	:	B.E	1	or . V

Branch - Stream : MECHANICAL Course Type / Code : THEORY/21ME51

Course Title: THEORY OF MACHINES Date: 2/1/2024

Duration : 60 Minutes Max Marks : 20

Note: Answer ONE full question from each Module.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Mark s
1(a)	4 differences between machine and mechanism - 4x1 4 mark	4
(b)	lower pair - 2marks higher pair - 2marks kinamtic pairs based on relative motion- 4marks	8
2(a)	i)mechanism- 1mark ii)inversion- 1mark iii)lower pair & Higher pair – 1mark iv)DOF- 1mark	4
(b)	procedure velocity analysis of 4 bar mechanisms- 4marks acceleration analysis of 4 bar mechanisms- 4marks	8
3(a)	static equilibrium -1 mark i) Body subjected to two forces –sketch 0.5 mark, explantion-1 mark ii) body subjected to three forces–sketch 0.5 mark, explantion-1 mark	4
(b)	FBD of all the movable links	4
4(a)	tangential component of acceleration- 2marks radial component of acceleration- 2marks	4
(ь)	4 bar mechanism ABCD, space diagram- I mark velocity of the crank-1 marks velocity diagram- Imarks angular velocity of the link BC and CD-1marks	4

Name & Signature of Course In charge



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Degree : B.E

MECHANICAL

Branch - Stream : Course Title :

THEORY OF MACHINES

Duration : 60 Minutes

USN

Semester: V

Course Type / Code: THEORY/21ME51

Date: 5/2/2024

Max Marks: 20

Note: Answer ONE full question from each Module.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Marks	со	K- Level
	Module-1			
1(a)	Make use of involute profile to develop an expression for length of path of contact.	4	CO3	K2
(b)	Each of two gears in a mesh has 48 teeth and a module of 8 mm. The teeth are of 20° involute profile. The arc of contact is 2.25 times the circular pitch. Solve for the addendum.		соз	кз
_	OR			
2(a)	Develop an expression for Law of Gearing with a neat sketch	4	CO3	К3
(b)	Two involute gears in a mesh have a module of 8 mm and a pressure angle of 20°. The larger gear has 57 while the pinion has 23 teeth. If the addenda on pinion and		C03	КЗ
_	Module-2			
3(a)	A four link mechanism is acted upon by forces as shown in the figure. Determine the torque T ₂ to be applied on link 2 to keep the mechanism in equilibrium. AD=50mm, AB=40mm, BC=100mm, Dc=75mm, DE=35mm Dc=75mm, DE=35mm	4	CO2	К3
(b)	Explain the terms in governor i)Height of the Governor ii)minimum and maximum equilibrium speed	4	CO4	K2
	OR	-1		
4(a)	Determine T ₂ to keep the mechanism shown in the figure in equilibrium, AC=70mm,AB=150mm, O2A=40mm	4	CO2	кз
(b)	Explain the terms in governor i)isochronous governor ii)Hunting	4	CO4	K2

Name & Signature of Course In charge CNA-NAGADHILLM Name & Signature of Module Coordinator

HOD

Principal



USN

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Degree B.E

Semester: Course Type / Code: THEORY/21ME51

Branch - Stream : Course Title:

MECHANICAL THEORY OF MACHINES

Date: 5/2/2024

Duration 60 Minutes Max Marks: 20

Note: Answer ONE full question from each Module. K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Marks
1(a)	Path of contact = path of approach + path of recess $CD = CP + PD - 2+2 Marks$ $(\sqrt{R_a^2 - R^2 \cos^2 \varphi} - R \sin \varphi) - 4(\sqrt{r_o^2 - r^2 \cos^2 \varphi} - r \sin \varphi)$	4
(b)	Solution $\varphi = 20^\circ$; $t = T = 48$; $m = 8$ mm; $R = r = \frac{mT}{2} = \frac{8 \times 48}{2} = 192$ mm; $R_a = r_a$ Are of contact = $2.25 \times \text{Circular pitch} = 2.25\pi \text{ m}$ $P_c = \pi D/T$ $= 2.25\pi \times 8 = 56.55$ mm Path of contact = $56.55 \times \cos 20^\circ = 53.14$ mm or $(\sqrt{R_a^2 - R^2 \cos^2 \varphi} - R \sin \varphi)$ $+(\sqrt{r_a^2 - r^2 \cos^2 \varphi} - r \sin \varphi) = 53.14$ or $2(\sqrt{R_a^2 - 192^2 \cos^2 20^\circ} - 192 \sin 20^\circ)$ $= 53.14$ or $R_a = 202.6$ mm Addendum = $R_a - R = 202.6 - 192 = 10.6$ mm	2+2
2(a)	sketch - 2MARKS expression for Law of Gearing-2MARKS	4
(b)	length of path of contact=52.3mm arc of contact =55.6mm contact ratio.=1.5	8

3(a)	To the state of th	Page 34 of 7
(b)	Each definition is of 2 marks	4
4(a)	Fize friends fried fries.	4
(b)	T ₂ = F ₃₂ *h • F ₁₂ *h F ₃₂ and F ₁₂ form a CCW couple and hence T ₂ aces dock wise. Each definition is of 2 marks	1 4

Name & Signature of Course In charge



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Degree : B.E

Branch - Stream : MECHANICAL

Course Title: TH

THEORY OF MACHINES

: 60 Minutes

USN					
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Semester: V

Course Type / Code: THEORY/21ME51

Date: 5/2/2024

Max Marks: 20

Note: Answer ONE full question from each Module.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Marks	со	K- Level
-	Module-1			
1(a)	Make use of involute profile to develop an expression for length of path of contact.	4	CO3	К3
(b)	Two involute gears in a mesh have a module of 8 mm and a pressure angle of 20°. The larger gear has 57 while the pinion has 23 teeth. If the addenda on pinion and gear wheels are equal to one module, solve for (i) contact ratio (ii) angle of action of the pinion and the gear wheel (iii) ratio of the sliding to rolling velocity at the(a) beginning of contact (b) pitch point (c) end of contact	8	CO3	К3
	OR			
2(a)	Compare Cycloidal and involute gear toot profile	4	CO3	К3
	A pinion having 30 teeth drives a gear having 80 teeth. The profile of the gears is involute with 20° pressure angle, 12 mm module and 10 mm addendum. Solve for length of path of contact, are of contact and the contact ratio.	8	CO3	КЗ
	Module-2&4			
3(a)	Figure shows a slider crank mechanism in which the resultant gas pressure 8 x 10 ⁴ Nm- ² acts on the piston of cross sectional area 0.1 m ² . The system is kept in equilibrium as a result of the couple applied to the crank 2, through the shaft at O ₂ . Determine forces acting on all the links (including the pins) and the couple on 2.	4	CO2	К3
(b)	Explain the terms in governor i)Height of the Governor ii)minimum and maximum equilibrium speed	4	CO4	K2
	OR			
4(a)	Determine the torque T ₂ required to keep the given mechanism in equilibrium. O ₂ A= 30mm, = AB = O ₄ B, O ₂ O ₄ = 60mm, A O ₂ O ₄ = 60°, BC = 19mm, AD=15mm T ₂ · ?	4 ×	CO2	КЗ
(b)	Explain the terms in governor :i))isochronous governor ii)Hunting	4	CO4	K2

Name & Signature of Course

Name & Signature of Module Coordinator 1000

Principal .

Duration

60 Minutes



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K.S. INSTITUTE OF TECHNOLOGY, BENGALURU - 560109 SECOND INTERNAL TEST QUESTION PAPER 2023-24 ODD SEMESTER

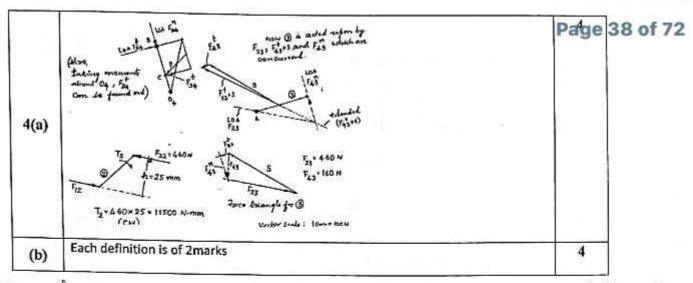
Max Marks: 20

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SET: B		.10	USN	
Degree	:	B.E	Semester : V	
Branch - Stream	:	MECHANICAL	Course Type / Code: THEORY/21M	E51
Course Title		THEORY OF MACHINES	Date: 5/2/2024	

Note: Answer ONE full question from each Module.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Marks
1(a)	Path of contact = path of approach + path of recess $CD = CP + PD-2+2 \text{ marks}$ $(\sqrt{R_{\sigma}^2 - R^2 \cos^2 \varphi} - R \sin \varphi)$	4
(b)	+ $(\sqrt{r_o^2 - r^2 \cos^2 \varphi} - r \sin \varphi)$. contact ratio = 1.68 (ii) angle of action of the pinion and the gear wheel= 10.63° (iii) ratio of the sliding to rolling velocity at the (a) beginning of contacT= = 0.32	8
	(b) pitch point=0 (c) end of contact=	4
2(a)	Cycloidal and involute gear toot profile	•
(Ŋ	length of path of contact-2 are of contact contact ratio.	8
3(a)	Figure 6.5 = 10 ³ N Long Feet Long Feet	4
(b)	Each definition is of 2marks	4



Name & Signature of Course In charge



SET: A

Degree :

B.E

Branch - Stream :

ME-ME

Course Title : Duration :

THEORY OF MACHINES

60 Minutes

USN

Semester: V

Course Type / Code:

Theory/21ME51

Date: 04.03.2024

Max Marks: 20

Note: Answer ONE full question from each part.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Questions	Marks	co	K- Level
PART-A			
Develop differential equation for a spring mass damper system	4	CO5	КЗ
A vibrating system consists of mass 25 kg, a spring of stiffness 15 kN/m and a damper. The damping provided is only 15% of critical value. Determine i) Critical damping coefficient ii) damping factor iii)natural frequency iv) logarithmic decrement v) ration of two consecutive amplitudes of vibration	8	CO5	КЗ
OR		Y	
A body of mass 5kg is supported on a spring of stiffness 1960N/m and has a dashpot connected to it which produces a resistance of 1.96N at a velocity of 1m/s in what ratio will the amplitude of vibration be reduced after 5 cycles	6	CO5	К3
Develop differential equation for a spring mass damper system and obtain complete solution for over damped system	6	CO5	кз
PART -B			
In an engine governor of porter type, the upper and lower arms are 200mm and 250mm respectively and pivoted on the axis of rotation. The mass of central load is 15kg, the mass of each ball is 2kg and friction at the sleeve together with the resistance of the operating gear is equal to a load of 24N at the sleeve. If the limiting inclinations of the upper arms to the verticals are 30° and 40°. Solve taking friction in to account range of speed of the governor.	8	CO4	кз
OR	0		
In a hartnell governor, the extreme radii of rotation of the balls are 40mm and 60mm and corresponding speeds are 210rpm and 230rpm. The mass of each ball is 3 kg. The ball and sleeve arms are equal. Determine i) spring loads at min and max speeds ii) stiffness of the spring iii)initial compression of the spring	8	CO4	к3
	Develop differential equation for a spring mass damper system A vibrating system consists of mass 25 kg, a spring of stiffness 15 kN/m and a damper. The damping provided is only 15% of critical value. Determine i) Critical damping coefficient ii) damping factor iii)natural frequency iv) logarithmic decrement v) ration of two consecutive amplitudes of vibration OR A body of mass 5kg is supported on a spring of stiffness 1960N/m and has a dashpot connected to it which produces a resistance of 1.96N at a velocity of Im/s in what ratio will the amplitude of vibration be reduced after 5 cycles Develop differential equation for a spring mass damper system and obtain complete solution for over damped system PART -B In an engine governor of porter type, the upper and lower arms are 200mm and 250mm respectively and pivoted on the axis of rotation. The mass of central load is 15kg, the mass of each ball is 2kg and friction at the sleeve together with the resistance of the operating gear is equal to a load of 24N at the sleeve, If the limiting inclinations of the upper arms to the verticals are 30° and 40°. Solve taking friction in to account range of speed of the governor. OR In a hartnell governor, the extreme radii of rotation of the balls are 40mm and 60mm and corresponding speeds are 210rpm and 230rpm. The mass of each ball is 3 kg. The ball and sleeve arms are equal. Determine i) spring loads at min and max speeds ii) stiffness of the spring iii)initial compression of the	PART-A Develop differential equation for a spring mass damper system A vibrating system consists of mass 25 kg, a spring of stiffness 15 kN/m and a damper. The damping provided is only 15% of critical value. Determine i) Critical damping coefficient ii) damping factor iii)natural frequency iv) logarithmic decrement v) ration of two consecutive amplitudes of vibration OR A body of mass 5kg is supported on a spring of stiffness 1960N/m and has a dashpot connected to it which produces a resistance of 1.96N at a velocity of 1m/s in what ratio will the amplitude of vibration be reduced after 5 cycles Develop differential equation for a spring mass damper system and obtain complete solution for over damped system PART -B In an engine governor of porter type, the upper and lower arms are 200mm and 250mm respectively and pivoted on the axis of rotation. The mass of central load is 15kg, the mass of each ball is 2kg and friction at the sleeve together with the resistance of the operating gear is equal to a load of 24N at the sleeve. If the limiting inclinations of the upper arms to the verticals are 30° and 40°. Solve taking friction in to account range of speed of the governor. OR In a hartnell governor, the extreme radii of rotation of the balls are 40mm and 60mm and corresponding speeds are 210rpm and 230rpm. The mass of each ball is 3 kg. The ball and sleeve arms are equal. Determine i) spring loads at min and max speeds ii) stiffness of the spring iii)initial compression of the	Develop differential equation for a spring mass damper system A vibrating system consists of mass 25 kg, a spring of stiffness 15 kN/m and a damper. The damping provided is only 15% of critical value. Determine i) Critical damping coefficient ii) damping factor iii)natural frequency iv) logarithmic decrement v) ration of two consecutive amplitudes of vibration OR A body of mass 5kg is supported on a spring of stiffness 1960N/m and has a dashpot connected to it which produces a resistance of 1.96N at a velocity of 1m/s in what ratio will the amplitude of vibration be reduced after 5 cycles Develop differential equation for a spring mass damper system and obtain complete solution for over damped system PART -B In an engine governor of porter type, the upper and lower arms are 200mm and 250mm respectively and pivoted on the axis of rotation. The mass of central load is 15kg, the mass of each ball is 2kg and friction at the sleeve together with the resistance of the operating gear is equal to a load of 24N at the sleeve. If the limiting inclinations of the upper arms to the verticals are 30° and 40°. Solve taking friction in to account range of speed of the governor. OR In a hartnell governor, the extreme radii of rotation of the balls are 40mm and 60mm and corresponding speeds are 210rpm and 230rpm. The mass of each ball is 3 kg. The ball and sleeve arms are equal. Determine i) spring loads at min and max speeds ii) stiffness of the spring iii)initial compression of the

Name & Signature of Course In charge: CM - NATAOW INTO Name & Signature of Module Coordinator:

HOD

Principal



SET: B

Branch - Stream :

Degree : B.E

D.E.

Course Title : Duration : ME-ME

THEORY OF MACHINES 60 Minutes USN

Semester : \

Course Type / Code:

: Theory/21ME51

Date: 04.03.2024

Max Marks: 20

Note: Answer ONE full question from each part.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Marks
1(a)	$x = e^{-\beta t} (A_1 + A_2 t)$	4
(b)	logarithmic decrement=0.6105-3Marks ratio of any two consecutive amplitudes of a vibrating system =1.84-3Mar number of cycles :2.63-2Marks	ks 8
2(a)	logarithmic decrement=0.63-3Marks ratio of any two consecutive amplitudes of a vibrating system =6.62-3Mar	ks 6
(b)	$x(t) = c_1 e^{-C + i\sqrt{1 - c^2}} \left[\omega_n t + c_2 e^{-C + i\sqrt{1 - c^2}} \right] + it$ $-e^{-C + i\sqrt{1 - c^2}} \left[z_1 \cos(\omega_n t) + d_2 \sin(\omega_n t) \right]$ $\Rightarrow x(t) = \frac{e^{-C + i\sqrt{1 - c^2}}}{\exp(-c + i\sqrt{1 - c^2})} \left[z_1 \cos(\omega_n t) + \frac{v_0 + c_{m_1} v_1}{\omega_n} \sin(\omega_n t) \right]$ From the formula in the following.	. 6
3(a)	N=167 rpm-2marks N1=164.8rpm-2marks N2=169rpm-2 marks N2-N1=4.2rpm- 2 marks	8.
4(a)	s1=562N- 2marks s2=930N- 2marks k=9.2N/mm-2 marks initial compression=9.2N/mm- 2marks	8

Name & Signature of Course In charge:



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Branch - Stream :

Degree

B.E

Course Title : Duration :

ME-ME

THEORY OF MACHINES 60 Minutes

USN

Theory/21ME51 Course Type / Code:

04.03.2024 Date :

Max Marks: 20

Note: Answer ONE full question from each part.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions	Marks	со	K- Level
	, PART-A		-	V.
1(a)	Develop differential equation for a spring mass damper system	4	CO5	K3
(b)	Find logarithmic decrement, ratio of any two consecutive amplitudes of a vibrating system and number of cycles after which the original amplitude is reduced to 20%. The vibrating system is defined by the following parameters m=3.5kg, k-2.5N/mm, c=0.018Ns/m	8	CO5	К3
	OR			
2(a)	A vibrating system in a vehicle is to be designed with the following parameters k=100N/m, c=2Ns/m, m=1kg. calculate the decrease in amplitude after 3 complete oscillations and frequency of oscillation	6	COS	К3
(b)	Develop differential equation for a spring mass damper system and obtain complete solution for over damped system	6	COS	кз
	PART -B			- 19
3(a)	The arms of a porter governor are 300mm long. The upper arms are pivoted on the axis of revolution. The lower arms are attached to the sleeve at a distance of 40mm from the axis of rotation. The mass of the load on the sleeve is 70kg and mass of each ball is 10kg. Find the equilibrium speed when the radius of rotation of the ball is 200mm. If the friction is equivalent to 20N at the sleeve, what will be the range of speed for this position?	8	CO4	кз
-5000-000	OR			(F-0
4(a)	In a spring loaded Hartnell type governor, the extreme radii of rotation of the balls are 80 mm and 120 mm. The ball arm and the sleeve arm of the bell crank lever are equal in length. The mass of each ball is 2 kg. If the speeds at the two extreme positions are 400 and 420 r.p.m., find: 1, the initial compression of the central spring, and 2, the spring constant.	8	CO4	кз

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Module Coordinator:

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SCHEME

Degree

Branch - Stream : ME-ME THEORY OF MACHINES

Course Title: Duration 60 Minutes USN

Semester:

Course Type / Code: Theory/21ME51

Date: 04.03.2024

Max Marks: 20

Note: Answer ONE full question from each part.

K-Levels: K1-Remebering, K2-Understanding, K3-Applying, K4-Analyzing, K5-Evaluating, K6-Creating

Q No.	Questions		Aarks
1(a)	$x = e^{-\beta t}(A_1 + A_2 t)$		4
7005	i) Given data-1Marks ii) Critical damping coefficient=1224.7Ns/m-1Marks iii) damping factor =0.15-1Marks		8
(b)	iv) natural frequency =24.22rad/s-1Marks v) logarithmic decrement =0.9533-1Marks vi) ration of two consecutive amplitudes of vibration=2.594-2Marks		3
2(a)	c=1.96Ns/m-2Marks cc=197.99Ns/m-2Marks logarithmic decrement:0.0622-2Marks x0/x5=1.365-2Marks		6
(b)	$z(t) = c_1 e^{\int_{-C}^{C} + t\sqrt{1 - C^2}} \omega_n t + c_2 e^{\int_{-C}^{C} + t\sqrt{1 - C^2}} \omega_n t $ $= e^{-\int_{-C}^{C} \omega_n t} \left[d_1 \cos(\omega_n t) + d_2 \sin(\omega_n t) \right]$ $\Rightarrow z(t) = \underbrace{e^{-\int_{-C}^{C} \omega_n t}}_{\text{Exponentially}} \left[z_0 \cos(\omega_n t) + \frac{v_0 + C\omega_n z_0}{\omega_n t} \sin(\omega_n t) \right]$ $= \frac{e^{-\int_{-C}^{C} \omega_n t}}{\text{Exponentially}} \left[z_0 \cos(\omega_n t) + \frac{v_0 + C\omega_n z_0}{\omega_n t} \sin(\omega_n t) \right]$ Function motion		6
3(a)	$N_1 = 183.3 \text{ r.p.m.}$ $N_2 = 222 \text{ r.p.m.}$ $N_2 - N_1 = 222 - 183.3 = 38.7 \text{ r.p.m.}$ Ans.	i t	8
4(a)	i) spring loads at min and max speeds i) stiffness of the spring-4 markd ii) iii)initial compression of the spring-4marks		8

Name & Signature of Course In charge: